

# Solving instabilities at small timesteps in CONTACT

Interim literature presentation

Niels van der Wekken

Applied Mathematics, TU Delft

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# VORtech

- ▶ Scientific Software Engineers
- ▶ 25 employees
- ▶ Delft, de Torenhove

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Distribution of CONTACT.

# CONTACT

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1978: DUVOROL

1982: CONTACT

1994: modernized

From here on many times extended and accelerated.

# Wiggles

Time discretization  $\Delta t$ ,  
rolling velocity  $V$ ,  
space discretization  $\Delta x$ .

## Definition

Traversed distance per timestep  $c = \frac{\Delta t \cdot V}{\Delta x}$ .



# Wiggles

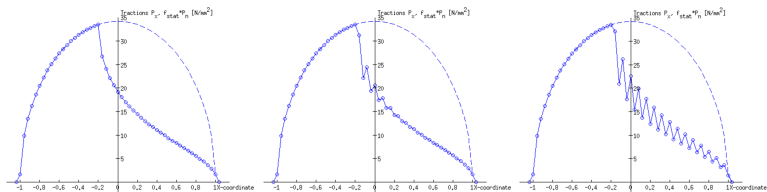
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## Definition

Traversed distance per timestep  $c = \frac{\Delta t \cdot V}{\Delta x}$ .

Instabilities arise when  $c \ll 1$ .

# Wiggles



(a) Case:  $c = 1$

(b) Case:  $c = 0.1$

(c) Case:  $c = 0.025$

Figure: The 2D Carter/Fromm problem using different timesteps.

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Particle in initial state has position  $\mathbf{x}$ .

Particle in deformed state has position  $\mathbf{x} + \mathbf{u}$ .

# Physics - strain

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Longitudinal strain:  $\epsilon_{ii} = \frac{\partial u_i}{\partial x_i}$ .

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Shear strain:  $\epsilon_{ij} = \frac{\partial u_j}{\partial x_i} + \frac{\partial u_i}{\partial x_j}, i \neq j$ .

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Strain is relative displacement caused by stretching and bending of the body.

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Link strain to stress through the generalized Hooke's Law with elastic tensor  $C$  (a material property).

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Simplification in an isotropic material:

$$\sigma_{ii} = \lambda(\epsilon_{11} + \epsilon_{22} + \epsilon_{33}) + 2\mu\epsilon_{ii}$$

and

$$\sigma_{ij} = \mu\epsilon_{ij}, i \neq j$$

with material properties  $\lambda$  and  $\mu$ .



# Situation

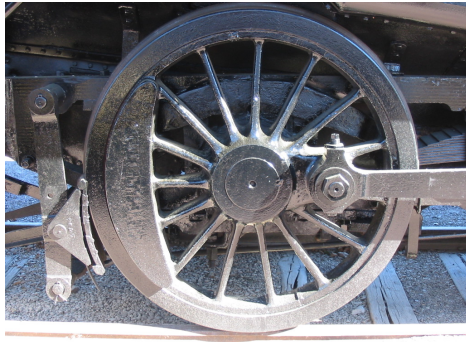


Figure: Train wheel on a rail.

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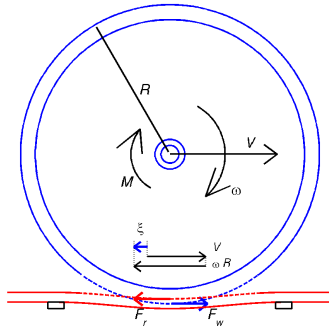


Figure: Schematic view of a wheel on a rail, where wheel and rail overlap there will be a contact area.

# Traction

Surface stress of body 1:  $\mathbf{p}^{(1)}$ , of body 2:  $\mathbf{p}^{(2)}$ .

Newton's third law of motion:

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Split the normal and tangential surface stress into:

- ▶ the scalar normal stress  $p_n$ ,
- ▶ the 2-vector tangential stress  $\mathbf{p}_\tau$ , the traction.

# Displacement

After deformation a point on the surface of body 1 has moved from  $\mathbf{x}^{(1)}$  to  $\mathbf{x}^{(1)} + \mathbf{u}^{(1)}$  by an amount  $\mathbf{u}^{(1)}$ , this is called the displacement.

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Before deformation the normal distance between the bodies is:

$$h = \mathbf{x}_n^{(1)} + \mathbf{x}_n^{(2)}.$$

After deformation this normal difference becomes

$$e = h - \mathbf{u}_n.$$



# Slip

## Definition

$$\text{Relative rigid slip } \mathbf{w} = \begin{bmatrix} \xi - \phi y \\ \eta + \phi x \end{bmatrix}.$$

A function of the overall longitudinal creepage  $\xi$ , lateral creepage  $\eta$ , and spin creepage  $\psi$ .

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## Definition

$$\text{Relative slip } \mathbf{s} = \mathbf{w} + \frac{\dot{\mathbf{u}}}{V}.$$

Both  $\mathbf{s}$  and  $\mathbf{w}$  are relative to the rolling velocity  $V$ .

# Contact conditions

In the normal problem:

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Coulombs friction law:  $g = \mu p_n$ .

# Influence function

Link between the tractions and displacements:

$$u_i(\mathbf{x}) = \int_C A_{ij}(\mathbf{x}, \mathbf{y}) p_j(\mathbf{y}) dS$$

# Boundary Element Methods

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## Disadvantages:

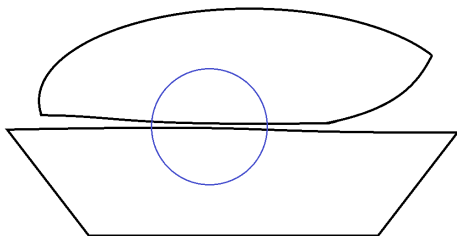
- When no Greens functions are known it is hard to find solutions in the interior.
- System of equations resulting from a BEM is usually dense.

## Half Space approach

Contact area small compared to radius of curvature at contact.

*“The contact stresses are highly concentrated close to the contact region and decrease rapidly in intensity with distance from the point of contact.”* - K.L. Johnson, 1985

Approximate the real solution by the solution of the contact between two half spaces.



## Quasi-identical behaviour

Simplification when the contacting bodies are geometrically and elastically symmetric.

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Simplification when the contacting bodies are geometrically and elastically symmetric.

Geometric symmetric: half-space.

Elastic symmetry: for modulus of rigidity  $G$  and Poisson's ratio  $\nu$  we must have

$$\frac{1 - 2\nu_1}{G_1} = \frac{1 - 2\nu_2}{G_2}.$$

## More simplifications

- ▶ Going from full 3D to 2D.
- ▶ Transient rolling vs steady-state.

# Discretization

Slip equations: relations between slip and displacements.

Contact conditions: relations between slip and tractions.

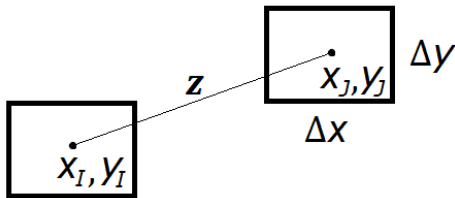
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$$A_{IiJj} = \int_{x_J - x_I - \frac{\Delta x}{2}}^{x_J - x_I + \frac{\Delta x}{2}} \int_{y_J - y_I - \frac{\Delta y}{2}}^{y_J - y_I + \frac{\Delta y}{2}} A_{ij}(\mathbf{z}) dz_2 dz_1$$

## Problem

In discretization a factor  $A(\mathbf{x}) - A'(\mathbf{x} + \mathbf{d}\mathbf{q})$  arises.

$c = \frac{dq}{dx}$  becomes small  $\rightarrow$  wiggles.

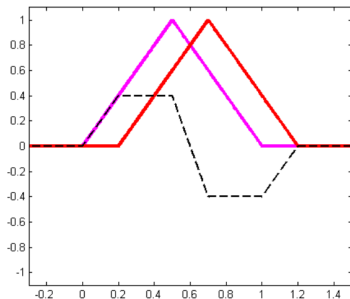
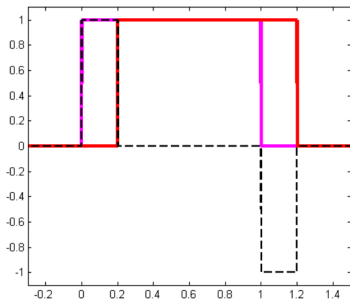


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Possible cause? Piecewise-approximation.



# Research Questions

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  - If this is not the case, how can we solve it?
3. How does replacing the piecewise constant basis functions by piecewise (bi)linear basis functions influence the rate of convergence of the algorithm?

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Questions?